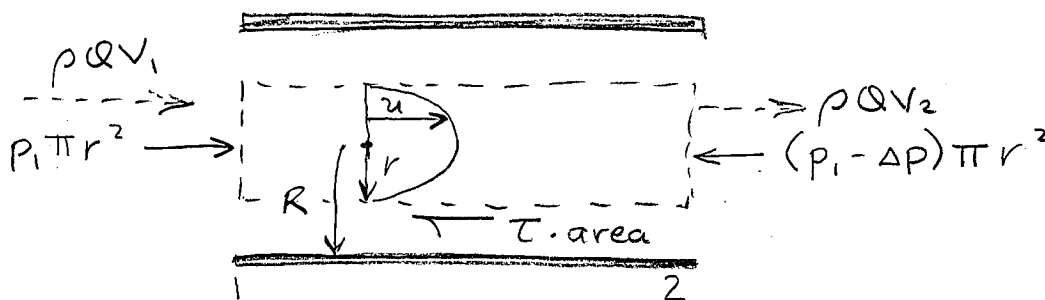
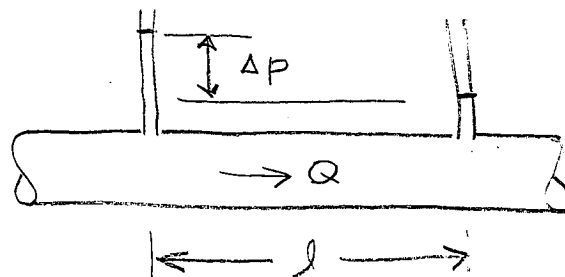


## DERIVATION OF HAGEN-POISEUILLE EQN

GIVEN: Laminar flow in a straight, horizontal pipe of radius  $R$ , and length  $l$

PROVE:  $Q = \frac{\pi D^4 \Delta p}{128 \mu l}$

SOLUTION:



assume:

- 1.  $\rho$  constant and incompressible
- 2.  $\perp$  to control surfaces
- 3. 2 control surfaces
- 4. fluid is NOT inviscid

$$\Rightarrow \sum F_x = \rho Q \left[ \int_{out} u(r) dA - \int_{in} u(r) dA \right]$$

BUT SINCE  $r_1 = r_2$  AND  $Q_1 = Q_2$ ,  $\bar{V}_1 = \bar{V}_2$ , SO

$$\sum F_x = 0!$$

$$\sum F = p_1 \pi r^2 - (p_1 - \Delta p) \pi r^2 - \tau \cdot \text{area}$$

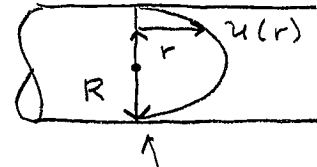
where area =

$$\Rightarrow \Delta p \pi r^2 = \tau (2\pi r l)$$

$$\Rightarrow \frac{\Delta p}{l} = \frac{2\tau}{r}$$



What we want to know



So... detour to look at  $\tau$  at the wall

since  $\frac{\Delta p}{l} \neq \phi(r)$ ,  $\Rightarrow \frac{2\tau}{r} \neq \phi(r)$  either

$$\Rightarrow \tau = \frac{\Delta p}{l} \frac{r}{2} = Cr \quad \text{where } C = \frac{\Delta p}{l \cdot 2} \quad (1)$$

Now looking at  $u(r)$ ,

When  $r = 0$ ,  $\tau = 0$

$r = R$ ,  $\tau = \tau_{\text{wall}} = \tau_{\text{max}}$

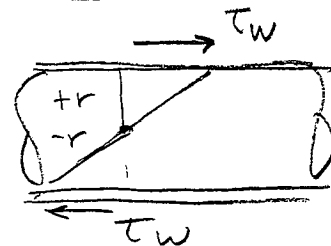


$$\Rightarrow \tau_w = CR = \frac{CD}{2} \quad \text{where } D = \text{diameter}$$

$$\text{and } C = \frac{2\tau_w}{D}$$

$$\text{plugging into (1), } \tau(r) = 2\tau_w \frac{r}{D}$$

$\tau$  distribution is linear...



What about velocity profile?

From  $\tau = \mu \frac{du}{dy}$ , for our case,  $dy = dr$

as  $r \uparrow$ ,  $u \downarrow$  so we have

$$\tau = -\mu \frac{du}{dr} \quad \text{to make } \tau \text{ positive}$$

from  $\frac{\Delta p}{l} = \frac{2\tau}{r}$ ,

and  $\frac{\Delta p}{l} = \frac{2}{r} \left( -\mu \frac{du}{dr} \right)$  NOW INTEGRATE!

$$\int du = - \int \frac{\Delta p}{l} \frac{1}{2\mu} r dr$$

$$u = \frac{-\Delta p r^2}{4\mu l} + C$$

To evaluate  $C$ , recognize that when

$$r = R = \frac{D}{2}, \quad u = 0$$

$$\Rightarrow 0 = \frac{-\Delta p R^2}{4\mu l} + C$$

$$\Rightarrow C = \frac{\Delta p}{4\mu l} \frac{D^2}{4} = \frac{\Delta p D^2}{16\mu l}$$

$$\text{and } u(r) = \frac{-\Delta p r^2}{4\mu l} + \frac{\Delta p D^2}{16\mu l} \quad (2)$$

now when  $r = 0$ ,  $u = u_{\max} = V_c$  (define)

$$V_c = \frac{\Delta p D^2}{16\mu l}$$

Work this into (2)

Take (2), reverse order on right-hand side:

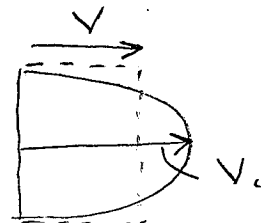
$$u(r) = \frac{\Delta p D^2}{16 \mu l} - \frac{\Delta p r^2}{4 \mu l} \cdot \underbrace{\frac{D^2}{D^2} \cdot \frac{4}{4}}_{\text{getting so we can use } V_c}$$

$$u(r) = \underbrace{\frac{\Delta p D^2}{16 \mu l}}_{V_c} - \underbrace{\frac{\Delta p D^2}{16 \mu l}}_{V_c} \left( \frac{2r}{D} \right)^2 \quad \text{"leftover"}$$

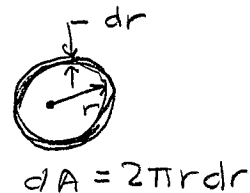
$$u(r) = V_c \left[ 1 - \left( \frac{2r}{D} \right)^2 \right] \quad \text{velocity distribution}$$

and average velocity?

$$Q = V \cdot A \quad \text{where } V = \text{avg velocity}$$



$$\begin{aligned} \int V dA &= \int u(r) dA \\ &= \int u(r) \cdot 2\pi r dr \end{aligned}$$



$$V \cdot \frac{\pi D^2}{4} = \int_0^R V_c \left[ 1 - \left( \frac{2r}{D} \right)^2 \right] 2\pi r dr$$

$$= 2\pi V_c \int_0^R \left[ 1 - \left( \frac{2r}{D} \right)^2 \right] r dr$$

$$= 2\pi V_c \left| \frac{r^2}{2} - \frac{4}{D^2} \frac{r^4}{4} \right|_0^R$$

$$V \cdot \frac{\pi D^2}{4} = 2\pi V_c \left( \frac{R^2}{2} - \frac{R^4}{D^2} \right)$$

$$\text{or } V \cdot \frac{\pi D^2}{4} = 2\pi V_c \left( \frac{D^2}{8} - \frac{D^2}{16} \right) \text{ from } R = \frac{D}{2}$$

$$V \frac{\pi D^2}{4} = 2\pi V_c \frac{D^2}{16}$$

$$\text{or } V = \frac{V_c}{2} !$$

$$\text{AND } Q = V \cdot \frac{\pi D^2}{4} = \frac{1}{2} (V_c) \frac{\pi D^2}{4}$$

$$= \frac{1}{2} \frac{\Delta p D^2}{16 \mu l} \cdot \frac{\pi D^2}{4}$$

$$Q = \frac{\pi D^4 \Delta p}{128 \mu l}$$

Q.E.D.

$$V = \frac{\Delta p D^2}{32 \mu l}$$

and considering flow at an angle  $\theta$



$$Q = \frac{\pi (\Delta p - \gamma l \sin \theta) D^4}{128 \mu l}$$

$$V = \frac{(\Delta p - \gamma l \sin \theta) D^2}{32 \mu l}$$